# The 'damping effect' in the dynamic response of stochastic systems

### S Adhikari

Zienkiewicz Centre for Computational Engineering, College of Engineering, Swansea University, Wales, UK S.Adhikari@swansea.ac.uk

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## Introduction

- 2 Stochastic damped SDOF systems
- The mean response of a dynamic system with uncertainty

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- Response in the time domain
- Response in the frequency domain
- Equivalent damping for the mean response

## Conclusions



Consider a normalised single degrees of freedom system (SDOF):

$$\ddot{u}(t) + 2\zeta \omega_n \, \dot{u}(t) + \omega_n^2 \, u(t) = f(t)/m \tag{1}$$

Here  $\omega_n = \sqrt{k/m}$  is the natural frequency and  $\xi = c/2\sqrt{km}$  is the damping ratio.

- We are interested in understanding the motion when the natural frequency of the system is perturbed in a stochastic manner.
- Stochastic perturbation can represent statistical scatter of measured values or a lack of knowledge regarding the natural frequency.



**Figure :** We assume that the mean of *r* is 1 and the standard deviation is  $\sigma_a$ .

Suppose the natural frequency is expressed as ω<sub>n</sub><sup>2</sup> = ω<sub>n0</sub><sup>2</sup> r, where ω<sub>n0</sub> is deterministic frequency and r is a random variable with a given probability distribution function.



Figure : 1000 sample realisations of the frequencies for the three distributions

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#### Response in the time domain



**Figure :** Response due to initial velocity  $v_0$  with 5% damping

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#### Response in the time domain



**Figure :** Response due to initial velocity  $v_0$  with 5% damping

#### **Frequency response function**



**Figure :** Normalised frequency response function with 5% damping  $|u/u_{st}|^2$ , where  $u_{st} = f/k$ 

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**Figure :** Normalised frequency response function with 5% damping  $|u/u_{st}|^2$ , where  $u_{st} = f/k$ 

- The mean response is significantly more damped compared to deterministic response 'The ghost damping'!
- The higher the randomness, the higher the "effective damping".
- The qualitative features are almost independent of the distribution the random natural frequency.
- We often use averaging to obtain more reliable experimental results is it always true?

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Assuming uniform random variable, we aim to explain some of these observations.

- Assume that the random natural frequencies are  $\omega_n^2 = \omega_{n_0}^2 (1 + \epsilon x)$ , where *x* has zero mean and unit standard deviation.
- The normalised harmonic response in the frequency domain

$$\frac{u(i\omega)}{f/k} = \frac{k/m}{[-\omega^2 + \omega_{n_0}^2(1+\epsilon x)] + 2i\xi\omega\omega_{n_0}\sqrt{1+\epsilon x}}$$
(2)

• Considering  $\omega_{n_0} = \sqrt{k/m}$  and frequency ratio  $r = \omega/\omega_{n_0}$  we have

$$\frac{u}{f/k} = \frac{1}{\left[(1+\epsilon x) - r^2\right] + 2i\xi r\sqrt{1+\epsilon x}}$$
(3)

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 The squared-amplitude of the normalised dynamic response can be obtained as

$$\left(\frac{|u|}{f/k}\right)^2 = \frac{1}{[(1+\epsilon x) - r^2]^2 + 4\xi^2 r^2 1 + \epsilon x}$$
(4)

We are interested in the response at the resonance. Therefore considering ω = ω<sub>n₀</sub> (that is r = 1), we have

$$\hat{U} = \left(\frac{|u|}{f/k}\right)^2|_{r=1} = \frac{1}{\epsilon^2 x^2 + 4\xi^2(1+\epsilon x)}$$
(5)

• Since x is zero mean unit standard deviation uniform random variable, its pdf is given by  $p_x(x) = 1/2\sqrt{3}, -\sqrt{3} \le x \le \sqrt{3}$ 

• The mean of  $\hat{U}$  is therefore

$$E\left[\hat{U}\right] = \int \frac{1}{\epsilon^{2}x^{2} + 4\xi^{2}(1+\epsilon x)} p_{x}(x) dx$$
  
$$= \frac{1}{2\sqrt{3}} \int_{-\sqrt{3}}^{\sqrt{3}} \frac{1}{\epsilon^{2}x^{2} + 4\xi^{2}(1+\epsilon x)} dx$$
  
$$= \frac{1}{4\sqrt{3}\epsilon\xi\sqrt{1-\xi^{2}}} \tan^{-1}\left(\frac{\sqrt{3}\epsilon}{2\xi\sqrt{1-\xi^{2}}} - \frac{\xi}{\sqrt{1-\xi^{2}}}\right)$$
  
$$+ \frac{1}{4\sqrt{3}\epsilon\xi\sqrt{1-\xi^{2}}} \tan^{-1}\left(\frac{\sqrt{3}\epsilon}{2\xi\sqrt{1-\xi^{2}}} + \frac{\xi}{\sqrt{1-\xi^{2}}}\right)$$
(6)

• Note that for any  $\delta$  we have

$$\frac{1}{2} \{ \tan^{-1}(a+\delta) + \tan^{-1}(a-\delta) \} = \tan^{-1}(a) + O(\delta^2)$$
(7)

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• Neglecting terms of the order  $O(\xi^2)$  we have

$$\operatorname{E}\left[\hat{U}\right] \approx \frac{1}{2\sqrt{3}\epsilon\xi\sqrt{1-\xi^2}} \tan^{-1}\left(\frac{\sqrt{3}\epsilon}{2\xi\sqrt{1-\xi^2}}\right) \approx \frac{\tan^{-1}(\sqrt{3}\epsilon/2\xi)}{2\sqrt{3}\epsilon\xi} \quad (8)$$

- For small damping, the maximum amplitude at ω = ω<sub>n₀</sub> is 1/4ξ<sup>2</sup><sub>e</sub> where ξ<sub>e</sub> is the equivalent damping for the mean response
- Therefore, the equivalent damping for the mean response is given by

$$(2\xi_{\theta})^{2} = \frac{2\sqrt{3}\epsilon\xi}{\tan^{-1}(\sqrt{3}\epsilon/2\xi)}$$
(9)

From this we can obtain the equivalent damping factor as

$$\xi_e = \frac{3^{1/4}\sqrt{\epsilon\xi}}{\sqrt{2\tan^{-1}(\sqrt{3}\epsilon/2\xi)}} \tag{10}$$

• For small damping and randomness, ignoring higher-order terms in  $\epsilon$  and  $\xi$  we obtain

$$\xi_e = \frac{3^{1/4}\sqrt{\epsilon\xi}}{\sqrt{2\tan^{-1}(\sqrt{3}\epsilon/2\xi)}} \approx \frac{3^{1/4}}{\sqrt{\pi}}\sqrt{\epsilon}\sqrt{\xi}$$
(11)

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• The equivalent damping factor of the mean system is proportional to the square root of the damping factor of the underlying baseline system



**Figure :** Normalised frequency response function with equivalent damping ( $\xi_e = 0.05$  in the ensembles). For the two cases  $\xi_e = 0.0551$  and  $\xi_e = 0.0643$  and respectively.



**Figure :** Normalised frequency response function with equivalent damping ( $\xi_e = 0.05$  in the ensembles). For the two cases  $\xi_e = 0.0735$  and  $\xi_e = 0.0819$  respectively.

- The mean response of a damped stochastic system is more damped than the underlying baseline system
- The 'damping effect' depends on the standard deviation of the resonance frequency the 'random' the system is, the more 'damped' the mean response becomes
- Any engineering design decision based on the mean result will NOT be a conservative one! This is because the 'sample response' can be significantly higher than the mean response.

- The mean response of a damped stochastic system is more damped than the underlying baseline system.
- For small baseline damping factors, the equivalent damping for the mean response is given by

$$\xi_e \approx rac{3^{1/4}\sqrt{\epsilon\xi}}{\sqrt{2\tan^{-1}(\sqrt{3}\epsilon/2\xi)}} \approx 3^{1/4}\sqrt{rac{\epsilon\xi}{\pi}}$$

where  $\epsilon$  is the standard deviation of the squared natural frequency and  $\xi$  is the damping factor of the baseline system.

- Higher standard deviation of the resonance frequency will 'dampen' the mean response.
- Any computational approach, based on perturbation around the baseline response, may have difficulty in predicting the statistical average when the damping of the baseline system is low.