# **Energy Harvesting Under Uncertainty**

S Adhikari<sup>1</sup>

<sup>1</sup>College of Engineering, Swansea University, Singleton Park, Swansea SA2 8PP, UK

IIT Madras, India

# **Swansea University**



# **Swansea University**



#### Collaborations

- Professor Mike Friswell and Dr Alexander Potrykus (Swansea University, UK).
- Professor Dan Inman (University of Michigan, USA)
- Professor Grzegorz Litak (University of Lublin, Poland).
- Professor Eric Jacquelin (University of Lyon, France)
- Professor S Narayanan and Dr S F Ali (IIT Madras, India).

**Funding:** Royal Society International Joint Project - 2010/R2: Energy Harvesting from Randomly Excited Nonlinear Oscillators (2 years from June 2011) - Swansea & IIT Madras (Prof Narayanan).

## Outline



#### Introduction

- Piezoelectric vibration energy harvesting
- The role of uncertainty

# Single Degree of Freedom Electromechanical Models

- Linear Systems
- Nonlinear System

# Optimal Energy Harvester Under Gaussian Excitation

- Circuit without an inductor
- Stochastic System Parameters
- Equivalent Linearisation Approach

# Conclusions

#### Piezoelectric vibration energy harvesting

- The harvesting of ambient vibration energy for use in powering low energy electronic devices has formed the focus of much recent research.
- Of the published results that focus on the piezoelectric effect as the transduction method, almost all have focused on harvesting using cantilever beams and on single frequency ambient energy, i.e., resonance based energy harvesting. Several authors have proposed methods to optimize the parameters of the system to maximize the harvested energy.
- Some authors have considered energy harvesting under wide band excitation.

#### Why uncertainty is important for energy harvesting?

- In the context of energy harvesting of ambient vibration, the input excitation may not be always known exactly.
- There may be uncertainties associated with the numerical values considered for various parameters of the harvester. This might arise, for example, due to the difference between the true values and the assumed values.
- If there are several nominally identical energy harvesters to be manufactured, there may be genuine parametric variability within the ensemble.
- Any deviations from the assumed excitation may result an optimally designed harvester to become sub-optimal.

# **Types of uncertainty**

Suppose the set of coupled equations for energy harvesting:

$$\mathcal{L}\{\mathbf{u}(t)\} = \mathbf{f}(t) \tag{1}$$

# Uncertainty in the input excitations

- For this case in general f(t) is a random function of time. Such functions are called random processes.
- **f**(*t*) can be stationary or non-stationary random processes

# Uncertainty in the system

- The operator  $\mathcal{L}$ {•} is in general a function of parameters  $\theta_1, \theta_2, \cdots, \theta_n \in \mathbb{R}$ .
- The uncertainty in the system can be characterised by the joint probability density function p<sub>Θ1,Θ2</sub>,...,<sub>Θn</sub> (θ1, θ2,..., θn).

#### **SDOF electromechanical models**



Schematic diagrams of piezoelectric energy harvesters with two different harvesting circuits. (a) Harvesting circuit without an inductor, (b) Harvesting circuit with an inductor.

For the harvesting circuit without an inductor, the coupled electromechanical behavior can be expressed by the linear ordinary differential equations

$$m\ddot{\mathbf{x}}(t) + c\dot{\mathbf{x}}(t) + k\mathbf{x}(t) - \theta\mathbf{v}(t) = f(t)$$
(2)

$$\theta \dot{x}(t) + C_p \dot{v}(t) + \frac{1}{R_l} v(t) = 0 \tag{3}$$

For the harvesting circuit with an inductor, the electrical equation becomes

$$heta\ddot{x}(t) + C_{\rho}\ddot{v}(t) + \frac{1}{R_{l}}\dot{v}(t) + \frac{1}{L}v(t) = 0$$
 (4)

#### Simplified piezomagnetoelastic model



Schematic of the

piezomagnetoelastic device. The beam system is also referred to as the 'Moon Beam'.

#### **Governing equations**

The nondimensional equations of motion for this system are

$$\ddot{x} + 2\zeta \dot{x} - \frac{1}{2}x(1 - x^2) - \chi v = f(t),$$
(5)

$$\dot{\boldsymbol{\nu}} + \lambda \boldsymbol{\nu} + \kappa \dot{\boldsymbol{x}} = \boldsymbol{0}, \tag{6}$$

where *x* is the dimensionless transverse displacement of the beam tip, *v* is the dimensionless voltage across the load resistor,  $\chi$  is the dimensionless piezoelectric coupling term in the mechanical equation,  $\kappa$  is the dimensionless piezoelectric coupling term in the electrical equation,  $\lambda \propto 1/R_IC_p$  is the reciprocal of the dimensionless time constant of the electrical circuit,  $R_I$  is the load resistance, and  $C_p$  is the capacitance of the piezoelectric material. The force f(t) is proportional to the base acceleration on the device. If we consider the inductor, then the second equation will be  $\ddot{v} + \lambda \dot{v} + \beta v + \kappa \ddot{x} = 0$ .

#### Possible physically realistic cases

Depending on the system and the excitation, several cases are possible:

- Linear system excited by harmonic excitation
- Linear system excited by stochastic excitation
- Linear stochastic system excited by harmonic/stochastic excitation
- Nonlinear system excited by harmonic excitation
- Nonlinear system excited by stochastic excitation
- Nonlinear stochastic system excited by harmonic/stochastic excitation

This talk is focused on application of random vibration theory to various energy harvesting problems

Our equations:

$$m\ddot{\mathbf{x}}(t) + c\dot{\mathbf{x}}(t) + k\mathbf{x}(t) - \theta\mathbf{v}(t) = -m\ddot{\mathbf{x}}_{b}(t)$$
(7)

$$\theta \dot{\mathbf{x}}(t) + C_{\rho} \dot{\mathbf{v}}(t) + \frac{1}{R_{I}} \mathbf{v}(t) = 0$$
(8)

Transforming both the equations into the frequency domain and dividing the first equation by m and the second equation by  $C_{\rho}$  we obtain

$$\left(-\omega^{2}+2i\omega\zeta\omega_{n}+\omega_{n}^{2}\right)X(\omega)-\frac{\theta}{m}V(\omega)=\omega^{2}X_{b}(\omega)$$
(9)  
$$i\omega\frac{\theta}{C_{p}}X(\omega)+\left(i\omega+\frac{1}{C_{p}R_{l}}\right)V(\omega)=0$$
(10)

The natural frequency of the harvester,  $\omega_n$ , and the damping factor,  $\zeta$ , are defined as

$$\omega_n = \sqrt{\frac{k}{m}}$$
 and  $\zeta = \frac{c}{2m\omega_n}$ . (11)

Dividing the preceding equations by  $\omega_n$  and writing in matrix form one has

$$\begin{bmatrix} (1 - \Omega^2) + 2i\Omega\zeta & -\frac{\theta}{k} \\ i\Omega\frac{\alpha\theta}{C_p} & (i\Omega\alpha + 1) \end{bmatrix} \begin{pmatrix} X \\ V \end{pmatrix} = \begin{pmatrix} \Omega^2 X_b \\ 0 \end{pmatrix}, \quad (12)$$

where the dimensionless frequency and dimensionless time constant are defined as

$$\Omega = \frac{\omega}{\omega_n}$$
 and  $\alpha = \omega_n C_p R_l.$  (13)

 $\alpha$  is the time constant of the first order electrical system, non-dimensionalized using the natural frequency of the mechanical system.

Inverting the coefficient matrix, the displacement and voltage in the frequency domain can be obtained as

$$\left\{ \begin{array}{c} X \\ V \end{array} \right\} = \frac{1}{\Delta_1} \left[ \begin{array}{c} (\mathrm{i}\Omega\alpha + 1) & \frac{\theta}{k} \\ -\mathrm{i}\Omega\frac{\alpha\theta}{C_p} & (1 - \Omega^2) + 2\mathrm{i}\Omega\zeta \end{array} \right] \left\{ \begin{array}{c} \Omega^2 X_b \\ 0 \end{array} \right\} = \left\{ \begin{array}{c} (\mathrm{i}\Omega\alpha + 1)\Omega^2 X_b/\Delta_1 \\ -\mathrm{i}\Omega^3\frac{\alpha\theta}{C_p} X_b/\Delta_1 \end{array} \right\},$$
(14)

where the determinant of the coefficient matrix is

$$\Delta_{1}(\mathrm{i}\Omega) = (\mathrm{i}\Omega)^{3}\alpha + (2\zeta\alpha + 1)(\mathrm{i}\Omega)^{2} + (\alpha + \kappa^{2}\alpha + 2\zeta)(\mathrm{i}\Omega) + 1 \quad (15)$$

and the non-dimensional electromechanical coupling coefficient is

$$\kappa^2 = \frac{\theta^2}{kC_p}.$$
 (16)

# Mean power

The average harvested power due to the white-noise base acceleration with a circuit without an inductor can be obtained as  $E\left[\widetilde{P}\right] = E\left[\frac{|V|^2}{(R_I\omega^4\Phi_{x_bx_b})}\right] = \frac{\pi m \alpha \kappa^2}{(2 \zeta \alpha^2 + \alpha)\kappa^2 + 4 \zeta^2 \alpha + (2 \alpha^2 + 2)\zeta}.$ 

 From Equation (14) we obtain the voltage in the frequency domain as

$$V = \frac{-\mathrm{i}\Omega^3 \frac{\alpha\theta}{C_p}}{\Delta_1(\mathrm{i}\Omega)} X_b. \tag{17}$$

• We are interested in the mean of the normalized harvested power when the base acceleration is Gaussian white noise, that is  $|V|^2/(R_l\omega^4\Phi_{x_bx_b})$ .

The spectral density of the acceleration  $\omega^4 \Phi_{x_b x_b}$  and is assumed to be constant. After some algebra, from Equation (17), the normalized power is

$$\widetilde{P} = \frac{|V|^2}{(R_l \omega^4 \Phi_{x_b x_b})} = \frac{k \alpha \kappa^2}{\omega_n^3} \frac{\Omega^2}{\Delta_1(i\Omega) \Delta_1^*(i\Omega)}.$$
(18)

Using linear stationary random vibration theory, the average normalized power can be obtained as

$$\mathbf{E}\left[\widetilde{\boldsymbol{P}}\right] = \mathbf{E}\left[\frac{|\boldsymbol{V}|^2}{(\boldsymbol{R}_{l}\omega^4 \Phi_{\boldsymbol{x}_b \boldsymbol{x}_b})}\right] = \frac{k\alpha\kappa^2}{\omega_n^3} \int_{-\infty}^{\infty} \frac{\Omega^2}{\Delta_1(\mathrm{i}\Omega)\Delta_1^*(\mathrm{i}\Omega)} \,\mathrm{d}\omega \qquad (19)$$

From Equation (15) observe that  $\Delta_1(i\Omega)$  is a third order polynomial in  $(i\Omega)$ . Noting that  $d\omega = \omega_n d\Omega$  and from Equation (15), the average harvested power can be obtained from Equation (19) as

$$\mathbf{E}\left[\widetilde{\boldsymbol{P}}\right] = \mathbf{E}\left[\frac{|\boldsymbol{V}|^2}{(\boldsymbol{R}_{\boldsymbol{I}}\omega^4 \Phi_{\boldsymbol{X}_{\boldsymbol{b}}}\boldsymbol{X}_{\boldsymbol{b}})}\right] = \boldsymbol{m}\alpha\kappa^2 \boldsymbol{I}^{(1)}$$
(20)

$$I^{(1)} = \int_{-\infty}^{\infty} \frac{\Omega^2}{\Delta_1(i\Omega)\Delta_1^*(i\Omega)} \,\mathrm{d}\Omega. \tag{21}$$

After some algebra, this integral can be evaluated as

$$I^{(1)} = \frac{\pi}{\alpha} \frac{\det \begin{bmatrix} 0 & 1 & 0 \\ -\alpha & \alpha + \kappa^2 \alpha + 2\zeta & 0 \\ 0 & -2\zeta \alpha - 1 & 1 \end{bmatrix}}{\det \begin{bmatrix} 2\zeta \alpha + 1 & -1 & 0 \\ -\alpha & \alpha + \kappa^2 \alpha + 2\zeta & 0 \\ 0 & -2\zeta \alpha - 1 & 1 \end{bmatrix}}$$
(22)

Combining this with Equation (20) we obtain the average harvested power due to white-noise base acceleration.

#### Normalised mean power: numerical illustration



The normalized mean power of a harvester without an inductor as a function of  $\alpha$  and  $\zeta$ , with  $\kappa = 0.6$ . Maximizing the average power with respect to  $\alpha$  gives the condition  $\alpha^2 (1 + \kappa^2) = 1$  or in terms of physical quantities  $R_L^2 C_P (kC_P + \theta^2) = m$ .

The electrical equation becomes

$$heta \ddot{x}(t) + C_{\rho} \ddot{v}(t) + \frac{1}{R_{l}} \dot{v}(t) + \frac{1}{L} v(t) = 0$$
 (23)

where *L* is the inductance of the circuit. Transforming equation (23) into the frequency domain and dividing by  $C_p \omega_n^2$  one has

$$-\Omega^{2}\frac{\theta}{C_{p}}X + \left(-\Omega^{2} + i\Omega\frac{1}{\alpha} + \frac{1}{\beta}\right)V = 0$$
(24)

where the second dimensionless constant is defined as

$$\beta = \omega_n^2 L C_p, \tag{25}$$

Two equations can be written in a matrix form as

$$\begin{bmatrix} (1-\Omega^2)+2\mathrm{i}\Omega\zeta & -\frac{\theta}{k} \\ -\Omega^2\frac{\alpha\beta\theta}{C\rho} & \alpha(1-\beta\Omega^2)+\mathrm{i}\Omega\beta \end{bmatrix} \begin{bmatrix} X \\ V \end{bmatrix} = \begin{bmatrix} \Omega^2 X_b \\ 0 \end{bmatrix}.$$
 (26)

Vibration Energy Harvesting Under Uncertain

Inverting the coefficient matrix, the displacement and voltage in the frequency domain can be obtained as

$$\begin{cases} X \\ V \end{cases} = \frac{1}{\Delta_2} \begin{bmatrix} \alpha (1 - \beta \Omega^2) + i\Omega \beta & \frac{\theta}{k} \\ \Omega^2 \frac{\alpha \beta \theta}{C \rho} & (1 - \Omega^2) + 2i\Omega \zeta \end{bmatrix} \begin{cases} \Omega^2 X_b \\ 0 \end{cases}$$
$$= \begin{cases} \left(\alpha (1 - \beta \Omega^2) + i\Omega \beta \right) \Omega^2 X_b / \Delta_2 \\ \Omega^4 \frac{\alpha \beta \theta}{C \rho} X_b / \Delta_2 \end{cases}$$
(27)

where the determinant of the coefficient matrix is

$$\Delta_{2}(i\Omega) = (i\Omega)^{4}\beta \alpha + (2\zeta\beta\alpha + \beta)(i\Omega)^{3} + (\beta\alpha + \alpha + 2\zeta\beta + \kappa^{2}\beta\alpha)(i\Omega)^{2} + (\beta + 2\zeta\alpha)(i\Omega) + \alpha.$$
(28)

#### Mean power

The average harvested power due to the white-noise base acceleration with a circuit with an inductor can be obtained as  $E\left[\widetilde{P}\right] = \frac{m\alpha\beta\kappa^2\pi(\beta+2\alpha\zeta)}{\beta(\beta+2\alpha\zeta)(1+2\alpha\zeta)(\alpha\kappa^2+2\zeta)+2\alpha^2\zeta(\beta-1)^2}.$ 

• This can be obtained in a very similar to the previous case.

#### Normalised mean power: numerical illustration



The normalized mean power of a harvester with an inductor as a function of  $\alpha$  and  $\beta$ , with  $\zeta = 0.1$  and  $\kappa = 0.6$ .

#### **Optimal parameter selection**



The normalized mean power of a harvester with an inductor as a function of  $\beta$  for  $\alpha = 0.6$ ,  $\zeta = 0.1$  and  $\kappa = 0.6$ . The \* corresponds to the optimal value of  $\beta(=1)$  for the maximum mean harvested power.

#### **Optimal parameter selection**



The normalized mean power of a harvester with an inductor as a function of  $\alpha$  for  $\beta = 1$ ,  $\zeta = 0.1$  and  $\kappa = 0.6$ . The \* corresponds to the optimal value of  $\alpha (= 1.667)$  for the maximum mean harvested power.

#### **Stochastic system parameters**

- Energy harvesting devices are expected to be produced in bulk quantities
- It is expected to have some parametric variability across the 'samples'
- How can we take this into account and optimally design the parameters?

The natural frequency of the harvester,  $\omega_n$ , and the damping factor,  $\zeta_n$ , are assumed to be random in nature and are defined as

$$\begin{aligned}
\omega_n &= \bar{\omega}_n \Psi_\omega \tag{29} \\
\zeta &= \bar{\zeta} \Psi_\zeta \tag{30}
\end{aligned}$$

where  $\Psi_{\omega}$  and  $\Psi_{\zeta}$  are the random parts of the natural frequency and damping coefficient.  $\bar{\omega}_n$  and  $\bar{\zeta}$  are the mean values of the natural frequency and damping coefficient.

#### Mean harvested power: Harmonic excitation

The average (mean) normalized power can be obtained as

$$E[P] = E\left[\frac{|V|^2}{(R_l\omega^4 X_b^2)}\right]$$
$$= \frac{\bar{k}\alpha\kappa^2\Omega^2}{\bar{\omega}_n^3} \int_{-\infty}^{\infty} \int_{-\infty}^{\infty} \frac{f_{\Psi_\omega}(x_1)f_{\Psi_\zeta}(x_2)}{\Delta_1(\mathrm{i}\Omega, x_1, x_2)\Delta_1^*(\mathrm{i}\Omega, x_1, x_2)} \,\mathrm{d}x_1\mathrm{d}x_2 \quad (31)$$

where

$$\Delta_{1}(i\Omega, \Psi_{\omega}, \Psi_{\zeta}) = (i\Omega)^{3}\alpha + (2\bar{\zeta}\alpha\Psi_{\omega}\Psi_{\zeta} + 1)(i\Omega)^{2} + (\alpha\Psi_{\omega}^{2} + \kappa^{2}\alpha + 2\bar{\zeta}\Psi_{\omega}\Psi_{\zeta})(i\Omega) + \Psi_{\omega}^{2} \quad (32)$$

The probability density functions (pdf) of  $\Psi_{\omega}$  and  $\Psi_{\zeta}$  are denoted by  $f_{\Psi_{\omega}}(x)$  and  $f_{\Psi_{\zeta}}(x)$  respectively.

#### The mean power



The mean power for various values of standard deviation in natural frequency with  $\bar{\omega}_n = 670.5 \,\mathrm{rad/s}, \Psi_{\zeta} = 1, \alpha = 0.8649, \kappa^2 = 0.1185.$ 

#### The mean power



The mean harvested power for various values of standard deviation of the natural frequency, normalised by the deterministic power ( $\bar{\omega}_n = 670.5 \,\mathrm{rad/s}, \Psi_{\zeta} = 1, \alpha = 0.8649, \kappa^2 = 0.1185$ ).

#### **Optimal parameter selection**

The optimal value of  $\alpha$ :

$$\alpha_{\rm opt}^2 \approx \frac{(c_1 + c_2 \sigma^2 + 3c_3 \sigma^4)}{(c_4 + c_5 \sigma^2 + 3c_6 \sigma^4)}$$
(33)

where

$$c_1 = 1 + (4\bar{\zeta}^2 - 2)\Omega^2 + \Omega^4, c_2 = 6 + (4\bar{\zeta}^2 - 2)\Omega^2, c_3 = 1,$$
 (34)

$$c_{4} = \left(1 + 2\kappa^{2} + \kappa^{4}\right)\Omega^{2} + \left(4\bar{\zeta}^{2} - 2 - 2\kappa^{2}\right)\Omega^{4} + \Omega^{6},$$
 (35)

$$\mathbf{c}_{5} = \left(2\kappa^{2} + 6\right)\Omega^{2} + \left(4\bar{\zeta}^{2} - 2\right)\Omega^{4}, \qquad \mathbf{c}_{6} = \Omega^{2}, \qquad (36)$$

and  $\sigma$  is the standard deviation in natural frequency.

#### **Optimal parameter selection**

The optimal value of  $\kappa$ :

$$\kappa_{\rm opt}^2 \approx \frac{1}{(\alpha \ \Omega)} \sqrt{(d_1 + d_2 \sigma^2 + d_3 \sigma^4)}$$
 (37)

where

$$d_{1} = 1 + \left(4\bar{\zeta}^{2} + \alpha^{2} - 2\right)\Omega^{2} + \left(4\bar{\zeta}^{2}\alpha^{2} - 2\alpha^{2} + 1\right)\Omega^{4} + \alpha^{2}\Omega^{6}$$
(38)  
$$d_{2} = 6 + \left(4\bar{\zeta}^{2} + 6\alpha^{2} - 2\right)\Omega^{2} + \left(4\bar{\zeta}^{2}\alpha^{2} - 2\alpha^{2}\right)\Omega^{4}$$
(39)

$$d_{2} = 0 + \left(4\zeta + 0\alpha - 2\right)M + \left(4\zeta \alpha - 2\alpha\right)M$$
(00)

$$a_3 = 3 + 3\alpha^2 \Omega^2 \tag{40}$$

#### **Nonlinear coupled equations**

$$\ddot{\mathbf{x}} + 2\zeta \dot{\mathbf{x}} + g(\mathbf{x}) - \chi \mathbf{v} = f(t) \tag{41}$$

$$\dot{\mathbf{v}} + \lambda \mathbf{v} + \kappa \dot{\mathbf{x}} = \mathbf{0},\tag{42}$$

The nonlinear stiffness is represented as  $g(x) = -\frac{1}{2}(x - x^3)$ . Assuming a non-zero mean random excitation (i.e.,  $f(t) = f_0(t) + m_f$ ) and a non-zero mean system response (i.e.,  $x(t) = x_0(t) + m_x$ ), the following equivalent linear system is considered,

$$\ddot{x}_0 + 2\zeta \dot{x}_0 + a_0 x_0 + b_0 - \chi v = f_0(t) + m_f$$
(43)

where  $f_0(t)$  and  $x_0(t)$  are zero mean random processes.  $m_f$  and  $m_x$  are the mean of the original processes f(t) and x(t) respectively.  $a_0$  and  $b_0$  are the constants to be determined with  $b_0 = m_f$  and  $a_0$  represents the square of the natural frequency of the linearized system  $\omega_{eq}^2$ .

## Linearised equations

We minimise the expectation of the error norm i.e.,

 $(\mathbb{E} [\epsilon^2], \text{with } \epsilon = g(x) - a_0 x_0 - b_0)$ . To determine the constants  $a_0$  and  $b_0$  in terms of the statistics of the response x, we take partial derivatives of the error norm *w.r.t.*  $a_0$  and  $b_0$  and equate them to zero individually.

$$\frac{\partial}{\partial a_0} \mathbf{E} \left[ \epsilon^2 \right] = \mathbf{E} \left[ g(\mathbf{x}) \mathbf{x}_0 \right] - a_0 \mathbf{E} \left[ \mathbf{x}_0^2 \right] - b_0 \mathbf{E} \left[ \mathbf{x}_0 \right]$$
(44)  
$$\frac{\partial}{\partial b_0} \mathbf{E} \left[ \epsilon^2 \right] = \mathbf{E} \left[ g(\mathbf{x}) \right] - a_0 \mathbf{E} \left[ \mathbf{x}_0 \right] - b_0$$
(45)

Equating (44) and (45) to zero, we get,

$$a_{0} = \frac{\mathrm{E}\left[g(x)x_{0}\right]}{\mathrm{E}\left[x_{0}^{2}\right]} = \frac{\mathrm{E}\left[g(x)x_{0}\right]}{\sigma_{x}^{2}}$$
(46)  
$$b_{0} = \mathrm{E}\left[g(x)\right] = m_{f}$$
(47)

#### Responses of the piezomagnetoelastic oscillator



Simulated responses of the piezomagnetoelastic oscillator in terms of the

standard deviations of displacement and voltage ( $\sigma_x$  and  $\sigma_v$ ) as the standard deviation of the random excitation  $\sigma_f$  varies. (a) gives the ratio of the displacement and excitation; (b) gives the ratio of the voltage and excitation; and (c) shows the variance of the voltage, which is proportional to the mean power.

Adhikari (Swansea)

Vibration Energy Harvesting Under Uncertaint

#### **Phase portraits**



(b)  $\sigma_f = 0.045$ , (c)  $\sigma_f = 0.065$ . Note that the increasing noise level overcomes the potential barrier resulting in a significant increase in the displacement *x*.

#### Voltage output



Voltage output due to Gaussian white noise ( $\zeta = 0.01$ ,  $\chi = 0.05$ , and  $\kappa = 0.5$  and  $\lambda = 0.01$ .

#### Voltage output



Voltage output due to Lévy noise ( $\zeta = 0.01$ ,  $\chi = 0.05$ , and  $\kappa = 0.5$  and  $\lambda = 0.01$ .

#### Inverted beam harvester



(a) Schematic diagram of inverted beam harvester, (b) a typical phase portrait of the tip mass.

# Energy harvesting from bridge vibration



(a) Schematic diagram of a beam with a moving point load, (b) The variation in the energy generated by the harvester located at L/3 with  $\alpha$  for a single vehicle traveling at different speeds, u

# **Energy harvesting DVA**



(a) Schematic diagram of energy harvesting dynamic vibration absorber attached to a single degree of freedom vibrating system, (b)Harvested power in  $mW/m^2$ for nondimensional coupling coefficient  $\kappa^2 = 0.3$ 

#### Summary of the results

- Vibration energy based piezoelectric and magnetopiezoelectric energy harvesters are expected to operate under a wide range of ambient environments. This talk considers energy harvesting of such systems under harmonic and random excitations.
- Optimal design parameters were obtained using the theory of linear random vibration
- Nonlinearity of the system can be exploited to scavenge more energy over wider operating conditions
- Uncertainty in the system parameters can have dramatic affect on energy harvesting. This should be taken into account for optimal design
- Stochastic jump process models can be used for the calculation of harvested power

#### Conclusions

#### **Further details**

1	Ali, S. F., Friswell, M. I. and Adhikari, S., "Analysis of energy harvesters for highway bridges", Journal of Intelligent
	Material Systems and Structures, 22[16] (2011), pp. 1929-1938.
	Learning E. Adhillori, C. and Friennell, M. L. "Disperiestric device for impact energy her verting". Smort Materials on



Litak, G., Borowiec, B., Friswell, M. I. and Adhikari, S., "Energy harvesting in a magnetopiezoelastic system driven by random excitations with uniform and Gaussian distributions", Journal of Theoretical and Applied Mechanics, 49[3] (2011), pp. 757-764.

Ali, S. F., Adhikari, S., Friswell, M. I. and Narayanan, S., "The analysis of piezomagnetoelastic energy harvesters under broadband random excitations", Journal of Applied Physics, 109[7] (2011), pp. 074904:1-8

Ali, S. F., Friswell, M. I. and Adhikari, S., "Piezoelectric energy harvesting with parametric uncertainty", Smart Materials and Structures, 19[10] (2010), pp. 105010:1-9.

Friswell, M. I. and Adhikari, S., "Sensor shape design for piezoelectric cantilever beams to harvest vibration energy", Journal of Applied Physics, 108[1] (2010), pp. 014901:1-6.



Litak, G., Friswell, M. I. and Adhikari, S., "Magnetopiezoelastic energy harvesting driven by random excitations", Applied Physics Letters, 96[5] (2010), pp. 214103:1-3.

Adhikari, S., Friswell, M. I. and Inman, D. J., "Piezoelectric energy harvesting from broadband random vibrations", Smart Materials and Structures, 18[11] (2009), pp. 115005:1-7. Under Review



Ali, S. F. and Adhikari, S., "Energy harvesting dynamic vibration absorbers".

Friswell, M. I., Ali, S. F., Adhikari, S., Lees, A.W., Bilgen, O. and Litak, G., "Nonlinear piezoelectric vibration energy harvesting from an inverted cantilever beam with tip mass".